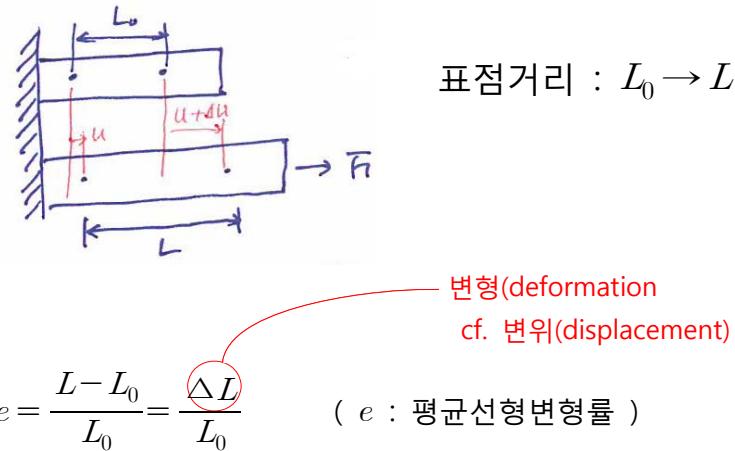


3장 변형률

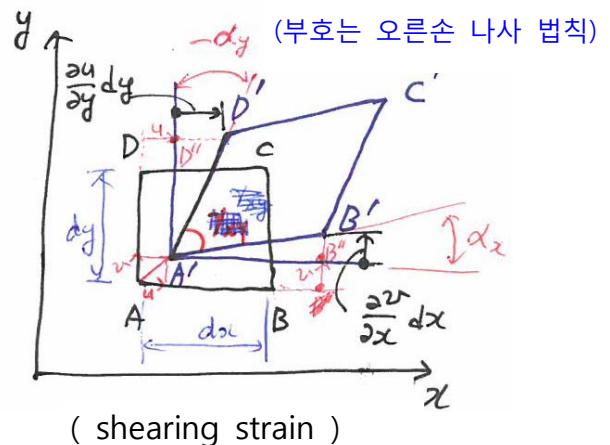
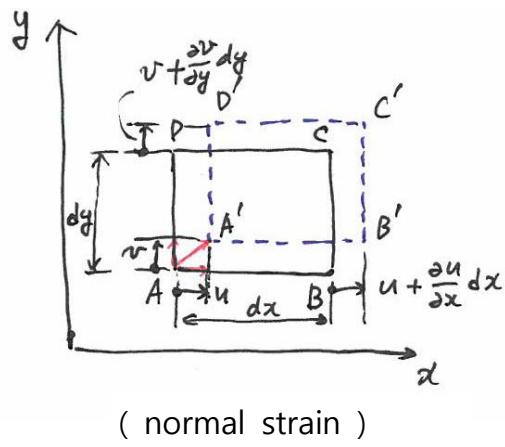
3.1 변위와 무한소변형률



- 한 점에서의 변형률: 표점거리가 0에 접근할 때 변형과 표점거리와의 비

$$\epsilon_x = \lim_{L \rightarrow 0} \frac{\Delta u}{L} = \frac{du}{dL} \quad \text{--- ①}$$

- 평면 변형률(plane strain) ; where in all points in the body, before and after application of load, remain in the same plane

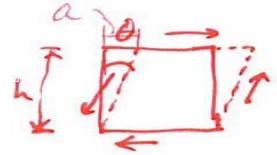


$$\epsilon_x = \frac{\partial u}{\partial x}, \quad \epsilon_y = \frac{\partial v}{\partial y}$$

공칭전단변형률 - 각의 변화량으로 나타냄

$$\Rightarrow \gamma = \frac{a}{h} = \tan\theta \approx \theta$$

- Now consider the change of angle DAB (in shearing strain)



- Assume the angle α_x : between AB and $A'B'$

$$\rightarrow \text{approximation } \alpha_x \approx \tan\alpha_x (= \frac{\partial v}{\partial x})$$

- Also, in view of the smallness of α_x , \rightarrow the normal strain is very small

$$\therefore AB \approx A'B''$$

$$\circ \text{ As } \alpha_x = \frac{\partial v}{\partial x} (\because \tan\alpha_x \approx \alpha_x = \frac{B''B}{A'B'} \approx \frac{B''B'}{AB} = \frac{\frac{\partial v}{\partial x}dx}{dx} = \frac{\partial v}{\partial x})$$

(counterclockwise rotation : +)

$$\rightarrow -\alpha_y = \frac{\partial u}{\partial y}$$

공칭전단변형률, 텐서량은 이것, 각도 변화량만 나타냄



- The total angular change of angle DAB, γ_{xy}

$$\gamma_{xy} = \alpha_x - \alpha_y = \frac{\partial v}{\partial x} + \frac{\partial u}{\partial y} = \frac{\partial u}{\partial y} + \frac{\partial v}{\partial x} \quad \text{--- ②}$$

\Rightarrow The shear strain is positive when the angle between two positive axes decrease.

- In the case of 3D element,

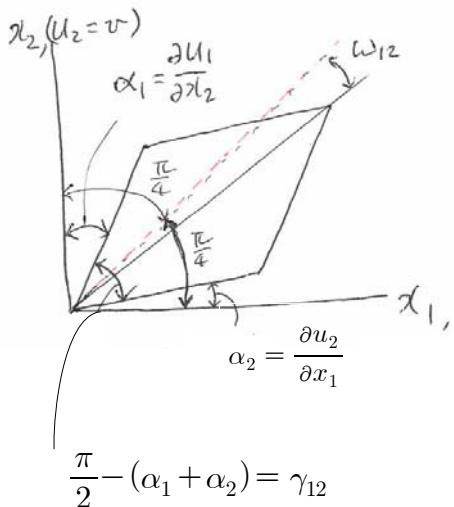
$$\epsilon_x = \frac{\partial u}{\partial x}, \quad \epsilon_y = \frac{\partial v}{\partial y}, \quad \epsilon_z = \frac{\partial w}{\partial z}$$

$$\gamma_{xy} = \frac{\partial u}{\partial y} + \frac{\partial v}{\partial x}, \quad \gamma_{yz} = \frac{\partial v}{\partial z} + \frac{\partial w}{\partial y}, \quad \gamma_{zx} = \frac{\partial w}{\partial x} + \frac{\partial u}{\partial z} \quad (\text{공칭전단변형률}) \quad \text{--- ③}$$

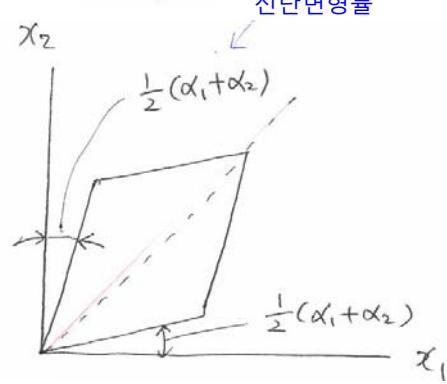
$$\rightarrow \gamma_{xy} = \gamma_{yx}, \quad \gamma_{yz} = \gamma_{zy}, \quad \gamma_{zx} = \gamma_{xz} \quad \text{--- ④}$$

- Eq ③ and ④ called to the strain-displacement relations of continuum mechanics

- 순수전단변형만을 고려해 보자



순수
회전



공칭전단변형률 γ_{12} 는 회전(순수회전)에 영향을 받지 않음

< 변형 후의 상태 >

< 순수전단 상태 >

- 따라서 전단변형률 $\epsilon_{12}(tensor) = \frac{1}{2}(\alpha_1 + \alpha_2) = \frac{1}{2}\left(\frac{\partial u_1}{\partial x_2} + \frac{\partial u_2}{\partial x_1}\right) = \frac{1}{2}\gamma_{12}$ 가 되고 $\epsilon_{21} = \epsilon_{12}$ 가 된다.

- 그러나 변형 후의 상태가 되기 위해서는 순수전단 상태를 ω_{12} 만큼 회전시켜야 한다.

$$\begin{aligned}\omega_{12} &= \frac{\pi}{4} - \left[\frac{1}{2}\left(\frac{\pi}{2} - (\alpha_1 + \alpha_2)\right) + \alpha_2 \right] \\ &= \frac{\pi}{4} - \left[\frac{\pi}{4} - \frac{\alpha_1 + \alpha_2}{2} + \alpha_2 \right] \\ &= \frac{1}{2}(\alpha_1 - \alpha_2) = \frac{1}{2}\left(\frac{\partial u_1}{\partial x_2} - \frac{\partial u_2}{\partial x_1}\right)\end{aligned}$$

- 일반화된 형태 $\omega_{ij} = \frac{1}{2}\left(\frac{\partial u_i}{\partial x_j} - \frac{\partial u_j}{\partial x_i}\right)$: 회전텐서

- Let $e_{11} = \frac{\partial u_1}{\partial x_1}, e_{12} = \frac{\partial u_1}{\partial x_2}, e_{ij} = \frac{\partial u_i}{\partial x_j}$ 로 두면,

$$\begin{aligned}e_{ij} &= \frac{1}{2}(e_{ij} + e_{ji}) + \frac{1}{2}(e_{ij} - e_{ji}) \\ &= \epsilon_{ij} + \omega_{ij}\end{aligned}$$

(e_{ij} : 상대변위텐서, ϵ_{ij} : 변형률텐서, ω_{ij} : 회전텐서)

$$e_{ij}(\text{상대변위텐서}) = \begin{bmatrix} e_{11} & e_{12} & e_{13} \\ e_{21} & e_{22} & e_{23} \\ e_{31} & e_{32} & e_{33} \end{bmatrix} = \begin{bmatrix} \frac{\partial u_1}{\partial x_1} & \frac{\partial u_1}{\partial x_2} & \frac{\partial u_1}{\partial x_3} \\ \frac{\partial u_2}{\partial x_1} & \frac{\partial u_2}{\partial x_2} & \frac{\partial u_2}{\partial x_3} \\ \frac{\partial u_3}{\partial x_1} & \frac{\partial u_3}{\partial x_2} & \frac{\partial u_3}{\partial x_3} \end{bmatrix}$$

→ 변위 u_i 는, $u_i = e_{ij}x_j$ 로 나타낸다.

$$\begin{aligned} u_1 &= e_{11}x_1 + e_{12}x_2 + e_{13}x_3 \\ u_2 &= e_{21}x_1 + e_{22}x_2 + e_{23}x_3 \\ u_3 &= e_{31}x_1 + e_{32}x_2 + e_{33}x_3 \end{aligned}$$

- $e_{ij} = \epsilon_{ij} + \omega_{ij}$ 이므로

$$u_i = \epsilon_{ij}x_j + \omega_{ij}x_j \not\models \text{됨}$$

here,

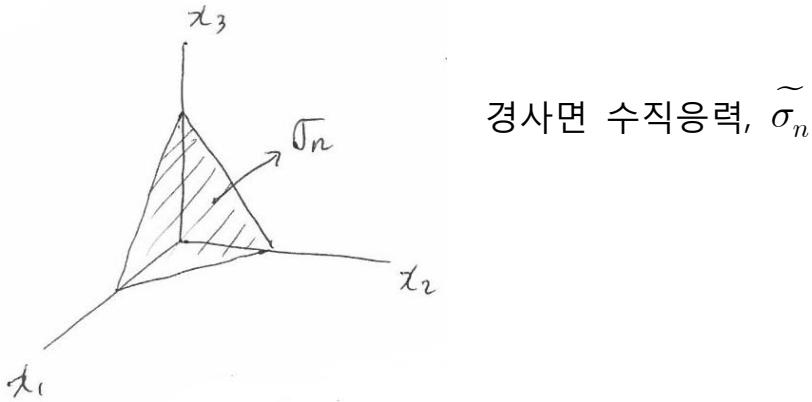
$$\epsilon_{ij} = \begin{bmatrix} \epsilon_{11} & \epsilon_{12} & \epsilon_{13} \\ \epsilon_{21} & \epsilon_{22} & \epsilon_{23} \\ \epsilon_{31} & \epsilon_{32} & \epsilon_{33} \end{bmatrix} = \begin{bmatrix} \frac{\partial u_1}{\partial x_1} & \frac{1}{2}\left(\frac{\partial u_1}{\partial x_2} + \frac{\partial u_2}{\partial x_1}\right) & \frac{1}{2}\left(\frac{\partial u_1}{\partial x_3} + \frac{\partial u_3}{\partial x_1}\right) \\ \frac{1}{2}\left(\frac{\partial u_2}{\partial x_1} + \frac{\partial u_1}{\partial x_2}\right) & \frac{\partial u_2}{\partial x_2} & \frac{1}{2}\left(\frac{\partial u_2}{\partial x_3} + \frac{\partial u_3}{\partial x_2}\right) \\ \frac{1}{2}\left(\frac{\partial u_3}{\partial x_2} + \frac{\partial u_1}{\partial x_3}\right) & \frac{1}{2}\left(\frac{\partial u_3}{\partial x_1} + \frac{\partial u_2}{\partial x_3}\right) & \frac{\partial u_3}{\partial x_3} \end{bmatrix}$$

$$(\epsilon_{ij} = \epsilon_{ji})$$

$$\omega_{ij} = \begin{bmatrix} \omega_{11} & \omega_{12} & \omega_{13} \\ \omega_{21} & \omega_{22} & \omega_{23} \\ \omega_{31} & \omega_{32} & \omega_{33} \end{bmatrix} = \begin{bmatrix} 0 & \frac{1}{2}\left(\frac{\partial u_1}{\partial x_2} - \frac{\partial u_2}{\partial x_1}\right) & \frac{1}{2}\left(\frac{\partial u_1}{\partial x_3} - \frac{\partial u_3}{\partial x_1}\right) \\ \frac{1}{2}\left(\frac{\partial u_2}{\partial x_1} - \frac{\partial u_1}{\partial x_2}\right) & 0 & \frac{1}{2}\left(\frac{\partial u_2}{\partial x_3} - \frac{\partial u_3}{\partial x_2}\right) \\ \frac{1}{2}\left(\frac{\partial u_3}{\partial x_2} - \frac{\partial u_1}{\partial x_3}\right) & \frac{1}{2}\left(\frac{\partial u_3}{\partial x_1} - \frac{\partial u_2}{\partial x_3}\right) & 0 \end{bmatrix}$$

$$(\omega_{ij} = -\omega_{ji})$$

- ϵ_{ij} 는 텐서량이므로 이미 설명한 응력텐서와 같은 성질을 가짐.
→ 응력에 대해 전개한 식에서 σ_{ij} 대신 ϵ_{ij} 를 대입하면 바로 성립
ex)



$$\tilde{\sigma}_n = \sigma_{11} l_1^2 + \sigma_{22} l_2^2 + \sigma_{33} l_3^2 + 2\sigma_{12} l_1 l_2 + 2\sigma_{23} l_2 l_3 + 2\sigma_{31} l_3 l_1$$

↓ 경사면 수직면에서 수직 변형률?

$$\begin{aligned} \tilde{\epsilon}_n &= \epsilon_{11} l_1^2 + \epsilon_{22} l_2^2 + \epsilon_{33} l_3^2 + 2\epsilon_{12} l_1 l_2 + 2\epsilon_{23} l_2 l_3 + 2\epsilon_{31} l_3 l_1 \\ &= \gamma_{12} \quad \quad \quad = \gamma_{23} \quad \quad \quad = \gamma_{31} \end{aligned}$$

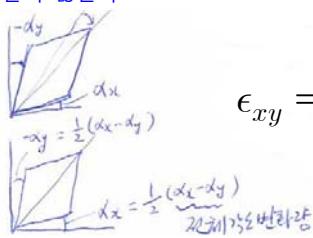
- A succinct statement of strain

$$\epsilon_{ij} = \frac{1}{2} \left(\frac{\partial u_i}{\partial x_j} + \frac{\partial u_j}{\partial x_i} \right) \quad i,j = x,y,z$$

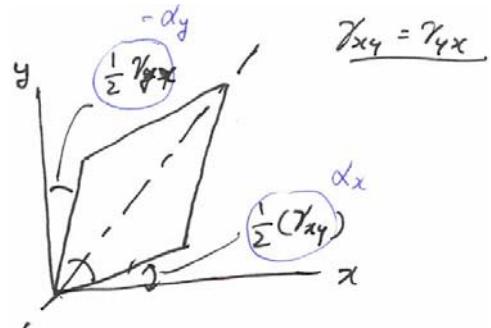
where $u_x = u$, $u_y = v$, $x_x = x$, $x_y = y$ and so on

* 전단변형률은 회전에 영향을 받지 않는다

ex) $i = x, j = y$



(ϵ_{xy}) : 변형률텐서, 응력텐서와 같은 성질, γ_{xy} : 전체 각도변화량, 공친 전단변형률)



$$\therefore \epsilon_{xy} = \frac{1}{2} \gamma_{xy}, \epsilon_{yz} = \frac{1}{2} \gamma_{yz}, \epsilon_{zx} = \frac{1}{2} \gamma_{zx} \quad \text{--- ⑤}$$

$$[\epsilon_{ij}] = \begin{bmatrix} \epsilon_x & \frac{1}{2}\gamma_{xy} & \frac{1}{2}\gamma_{xz} \\ \frac{1}{2}\gamma_{yx} & \epsilon_y & \frac{1}{2}\gamma_{yz} \\ \frac{1}{2}\gamma_{zx} & \frac{1}{2}\gamma_{zy} & \epsilon_z \end{bmatrix} \quad 9개 \rightarrow 6개 \quad \text{--- ⑥}$$

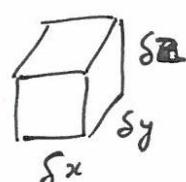
- 체적 변형률, Δ (delta), dilatation

: 체적 증가를 최초의 체적으로 나눈 값

- 최초의 체적 : $\delta x \delta y \delta z$

- 변형후 체적(신장 또는 압축에만 기인한다고 가정)

$$\begin{aligned} &: \delta x(1 + \epsilon_x)\delta y(1 + \epsilon_y)\delta z(1 + \epsilon_z) \\ &= \delta x \delta y \delta z (1 + \epsilon_x + \epsilon_y + \epsilon_z + \epsilon_x \epsilon_y + \epsilon_x \epsilon_z + \epsilon_y \epsilon_z + \epsilon_x \epsilon_y \epsilon_z) \end{aligned}$$



$$\therefore \delta V = \delta x \delta y \delta z (\epsilon_x + \epsilon_y + \epsilon_z)$$

$$\begin{aligned}\Delta &= \frac{\delta V}{V} = \frac{1}{\delta x \delta y \delta z} (\delta x \delta y \delta z (\epsilon_x + \epsilon_y + \epsilon_z)) \\ &= \epsilon_x + \epsilon_y + \epsilon_z = \frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} + \frac{\partial w}{\partial z}\end{aligned}\quad \text{—— ⑦}$$

* 변형의 적합조건식(strain compatibility equation)

: 6개의 변형률 성분은 3개의 변위 성분과 관계가 있기 때문에 변형률 성분 사이에 어떤 관계가 존재하지 않으면 안 된다.
이것을 '적합조건식'이라 한다.

⇒ Physically, this means that the body must be pieced together;
no voids are created in the deformed body

- In two-dimensional strain, we now proceed to develop the equations of compatibility, which establish the geometrically possible form of variation of strains from point to point within a body.
- Differentiation of ϵ_x twice with respect to y ,
 ϵ_y twice with respect to x ,
and γ_{xy} with respect to x and y

$$\frac{\partial^2 \epsilon_x}{\partial y^2} = \frac{\partial^2}{\partial y^2} \left(\frac{\partial u}{\partial x} \right) = \frac{\partial^3 u}{\partial x \partial y^2},$$

$$\frac{\partial^2 \epsilon_y}{\partial x^2} = \frac{\partial^3 v}{\partial x^2 \partial y},$$

$$\frac{\partial^2 \gamma_{xy}}{\partial x \partial y} = \frac{\partial^3 u}{\partial x \partial y^2} + \frac{\partial^3 v}{\partial x^2 \partial y}$$

$$\therefore \frac{\partial^2 \epsilon_x}{\partial y^2} + \frac{\partial^2 \epsilon_y}{\partial x^2} = \frac{\partial^2 \gamma_{xy}}{\partial x \partial y} \quad \text{--- ⑧}$$

변위: 3개 (u, v, w)

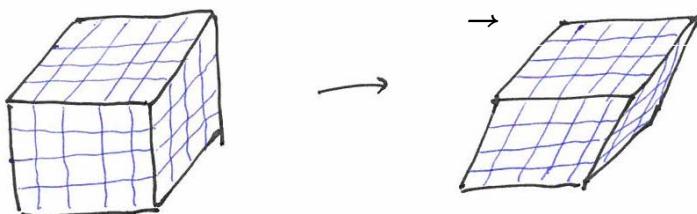
변형률: 6개 ($\epsilon_x, \epsilon_y, \epsilon_z, \gamma_{xy}, \gamma_{xz}, \gamma_{yz}$) (※ 변형률 성분들은 서로 독립적일 수가 없다)

- In three-dimensional equation of compatibility, they can be obtained by the same manner.

$$\begin{aligned} \frac{\partial^2 \epsilon_x}{\partial y^2} + \frac{\partial^2 \epsilon_y}{\partial x^2} &= \frac{\partial^2 \gamma_{xy}}{\partial x \partial y}, \quad 2 \frac{\partial^2 \epsilon_x}{\partial y \partial z} = \frac{\partial}{\partial x} \left(-\frac{\partial \gamma_{yz}}{\partial x} + \frac{\partial \gamma_{xz}}{\partial y} + \frac{\partial \gamma_{xy}}{\partial z} \right) \\ \frac{\partial^2 \epsilon_y}{\partial z^2} + \frac{\partial^2 \epsilon_z}{\partial y^2} &= \frac{\partial^2 \gamma_{yz}}{\partial y \partial z}, \quad 2 \frac{\partial^2 \epsilon_y}{\partial z \partial x} = \frac{\partial}{\partial y} \left(\frac{\partial \gamma_{yz}}{\partial x} - \frac{\partial \gamma_{xz}}{\partial y} + \frac{\partial \gamma_{xy}}{\partial z} \right) \\ \frac{\partial^2 \epsilon_z}{\partial x^2} + \frac{\partial^2 \epsilon_x}{\partial z^2} &= \frac{\partial^2 \gamma_{xz}}{\partial z \partial x}, \quad 2 \frac{\partial^2 \epsilon_z}{\partial x \partial y} = \frac{\partial}{\partial z} \left(\frac{\partial \gamma_{yz}}{\partial x} + \frac{\partial \gamma_{xz}}{\partial y} - \frac{\partial \gamma_{xy}}{\partial z} \right) \end{aligned} \quad \text{--- ⑨}$$

→ These equations were firstly derived by Saint-Venant in 1860.

Force



Prior to deformation

after deformation

⇒ 변형률 적합조건식이 만족하지 않으면 위와 같은 규칙적인 변형이 발생되지 못함.