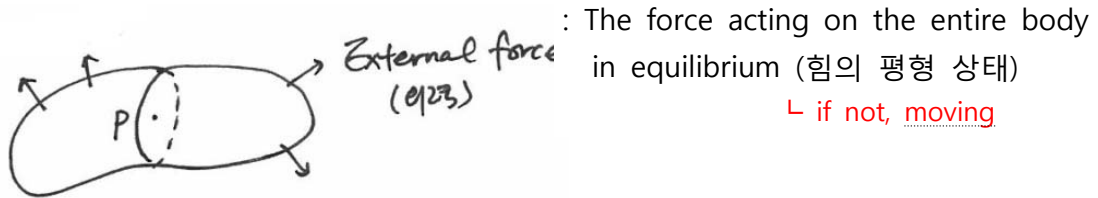
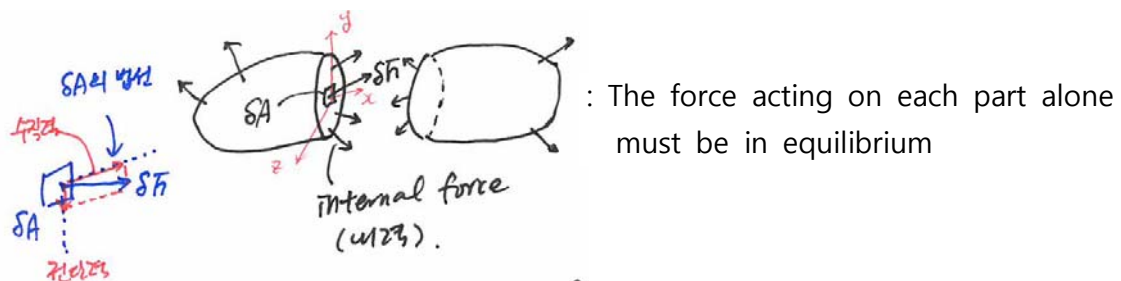


2장 응력(Stress)

2.1 응력



↳ dividing the body into two parts
using an imaginary plane

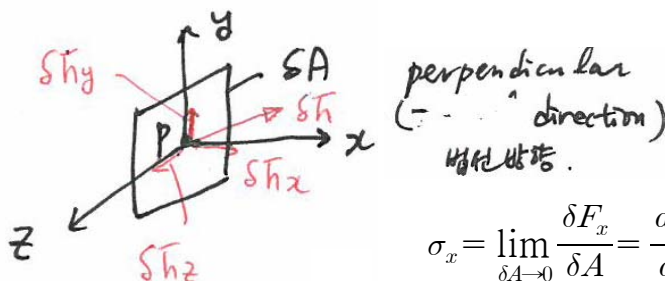


면적 δA 에서 평균응력(average stress) 개념

- Definition of stress : $\lim_{\delta A \rightarrow 0} \left(\frac{\delta F}{\delta A} \right)$ 가 어떤 값(Stress, 응력)으로 결정됨

→ P점에서의 미소면적(δA)을 지나고 P점에서 δF 의 방향을 가지는 상태
⇒ 따라서 응력은 특정한 평면과 그 평면에서의 특정 방향의 힘(응력 벡터)이 동시에 정의됨 (2nd order tensor)

↳ matrix



cf) Pressure(압력)

$$\sigma_x = \lim_{\delta A \rightarrow 0} \frac{\delta F_x}{\delta A} = \frac{dF_x}{dA} \quad : \text{normal stress}$$

(인장:+ 압축:-)

$$\tau_{xy} = \lim_{\delta A \rightarrow 0} \frac{\delta F_y}{\delta A} = \frac{dF_y}{dA}$$

$$\tau_{xz} = \lim_{\delta A \rightarrow 0} \frac{\delta F_z}{\delta A} = \frac{dF_z}{dA}$$

shear stress

※ Tensor(multi - dimensional array of numerical values)

Tensors, defined mathematically, are simply arrays of numbers, that transform according to certain rules under a change of coordinates. In physics, tensors characterize the properties of a physical system.

(ex, dot product, cross product, linear mapping)

⇒ 물리 현상을 기술하기 위하여 동비한 좌표계로 무한공간 또는 물리적 특성을 명확히 나타내기 위해 만들어진 일반화된 좌표계

rank, degree : dimensionality of the array needed to represent in

- Tensor of order zero (Scalar)

ex) mass of a particle → 작용점에서 하나의 값으로 표현

- Tensor of order one (Vector)

ex) force → 방향, 힘이 작용하는 점(위치) 모두 필요

- Tensor of order two (tensor) → matrix

ex) stress → 힘이 작용하는 점(면적), 그 면의 방향, 힘의 방향

$$\begin{bmatrix} \sigma_{xx} & \sigma_{xy} \\ \sigma_{yx} & \sigma_{yy} \end{bmatrix}$$

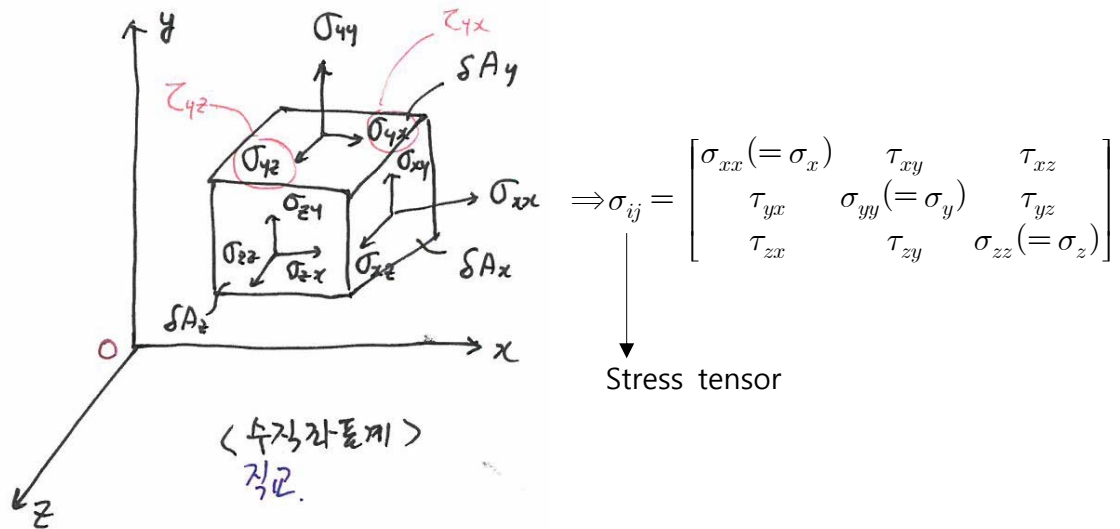
- Tensor of high order

Q. E-value는 어떤 tensor인가?

- 응력 성분

: 3차원 공간에서 한 점에 대한 완전한 응력 상태 표현 → 9개의 응력성분

↳ 한점을 지나며 서로 직교하는 3개의 평면에 관한
응력 성분을 모두 표기



cf). 원통좌표계(r, θ, z)인 경우 응력성분 → (P.21, 그림 2.5 참조)

- Unit(units of force per unit area)

- $\frac{N}{m^2} = Pa$ (Pascal) (SI) $\Rightarrow 1Pa = 0.000145 \text{ psi}$
- Pounds per square inch : psi (U.S)

- Equality of shear stress

* Equilibrium in all directions,

$$\Sigma F_x = 0, \Sigma F_y = 0, \Sigma F_z = 0$$

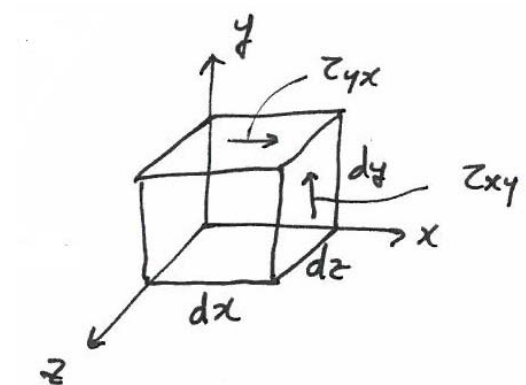
* Rotational equilibrium,

$$\Sigma M_x = 0, \Sigma M_y = 0, \Sigma M_z = 0$$

ex) $\Sigma M_z = 0,$

$$(-\tau_{xy} dy dz) dx + (\tau_{yx} dx dz) dy = 0$$

$$\therefore \tau_{xy} = \tau_{yx}$$



likewise, from $\Sigma M_x = 0, \Sigma M_y = 0$

$$\therefore \tau_{xz} = \tau_{zx}, \tau_{yz} = \tau_{zy}$$

→ This means that shearing stresses on mutually perpendicular planes of the element are equal.

$$\begin{bmatrix} \sigma_x & \tau_{xy} & \tau_{xz} \\ \tau_{yx} & \sigma_y & \tau_{yz} \\ \tau_{zx} & \tau_{zy} & \sigma_z \end{bmatrix} \longrightarrow \text{6개의 성분만 알면 된다}$$

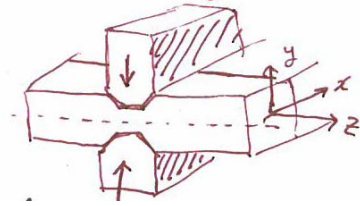
- Various types of stress condition

a. Triaxial stress

$$\begin{bmatrix} \sigma_1 & 0 & 0 \\ 0 & \sigma_2 & 0 \\ 0 & 0 & \sigma_3 \end{bmatrix}$$

: 주응력 상태, principal stress state

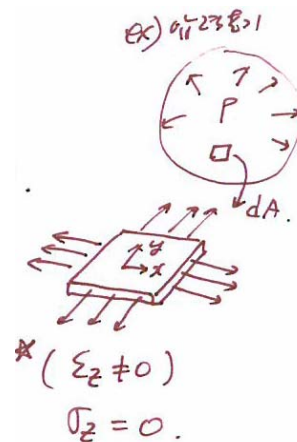
cf. plane strain $\begin{cases} \epsilon_z = 0 \\ \sigma_z = \nu(\sigma_x + \sigma_y) \end{cases}$



b. Two-dimensional or Plane stress

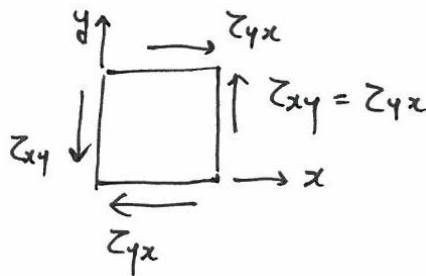
: 응력이 x, y 평면에만 작용하는 경우, (z 성분 = 0)

$$\begin{bmatrix} \sigma_x & \tau_{xy} & \tau_{xz} \\ \tau_{yx} & \sigma_y & \tau_{yz} \\ \tau_{zx} & \tau_{zy} & \sigma_z \end{bmatrix} \rightarrow \begin{bmatrix} \sigma_x & \tau_{xy} \\ \tau_{yx} & \sigma_y \end{bmatrix}$$



c. Pure shear

: In this case, the element is subjected to plane shearing stress only,

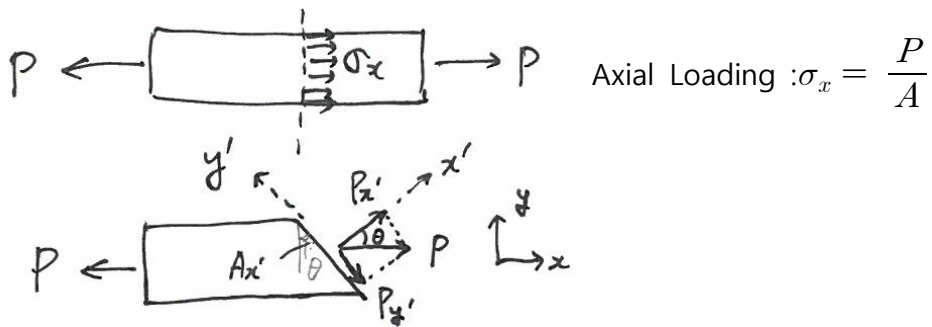


d. Uniaxial Stress

: normal stress act along one direction only,

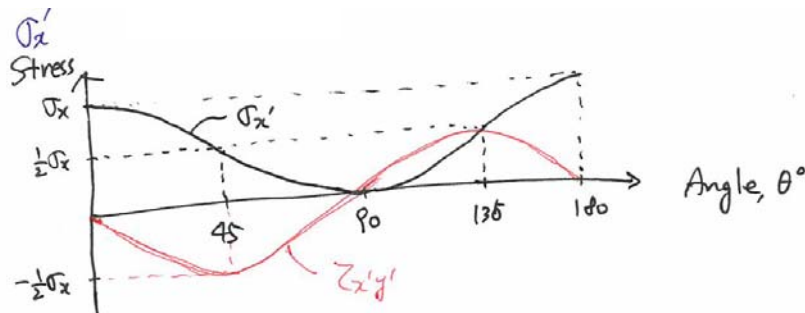
ex) uniaxial tension or compression

- Stresses on inclined planes in an axially loaded member



$$\sigma_{x'} = \frac{P \cos \theta (= P_{x'})}{A_{x'}} = \sigma_x \cos^2 \theta \quad (\leftarrow A_{x'} \cos \theta = \frac{A_x}{\cos \theta} \cdot \cos \theta = A_x) \quad - (*)$$

$$\tau_{x'y'} = -\frac{P \sin \theta}{A_{x'}} = -\sigma_x \sin \theta \cos \theta \quad \text{---} (**)$$

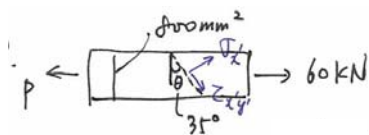


⇒ the normal stress is either maximum or a minimum on planes for which the shear stress is zero.

⇒ Shear stress in the range of $-\frac{1}{2}\sigma_x \sim \frac{1}{2}\sigma_x$

⇒ When $\theta > 90^\circ$, the sign of $\tau_{x'y'}$ is changed. (from - → to +)

Example)



Sol) $\sigma_x = \frac{P}{A} = \frac{60(10^3)}{800(10^{-6})} = 75 \text{ MPa}$

from (*), (**)

$$\therefore \sigma_{x'} = \sigma_x \cos^2 \theta = 50.33 \text{ MPa}, \tau_{x'y'} = -35.24 \text{ MPa}$$

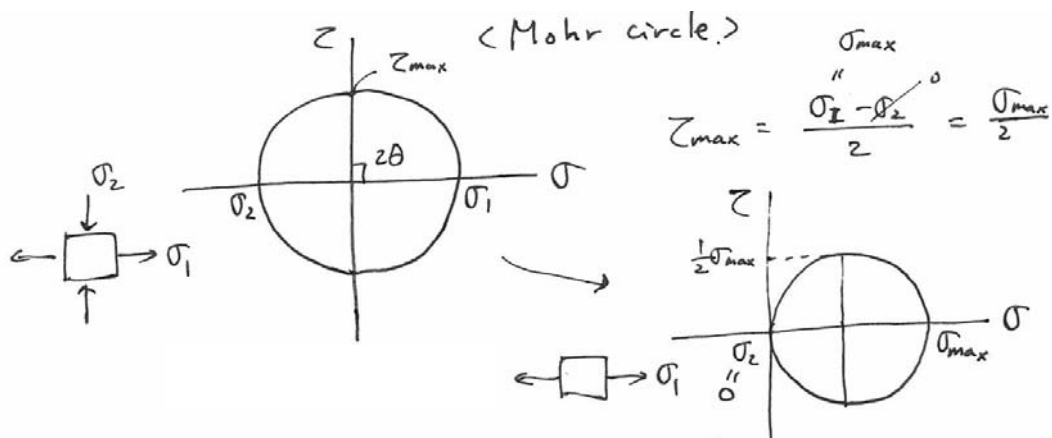
35° 평면에서 응력 상태는?

$\sigma_{x'}, \tau_{x'y'}$?

- From Eq(*) and (**),

As expected, $\sigma_{x'}$ is a maximum ($\sigma_{\max}$) when θ is 0° or 180° ,

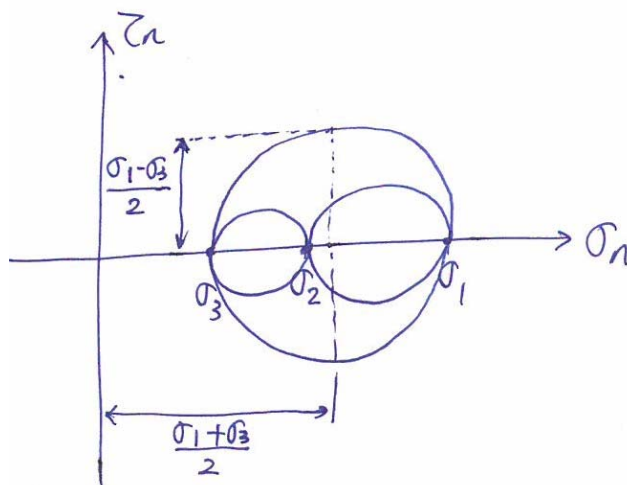
and $\tau_{x'y'}$ is maximum ($\tau_{\max}$) when θ is 45° or 135° , $\tau_{\max} = \pm \frac{1}{2}\sigma_{\max}$



<Mohr's circle>

* Mohr's circle is a two-dimensional graphical representation of the state of stress at a point.

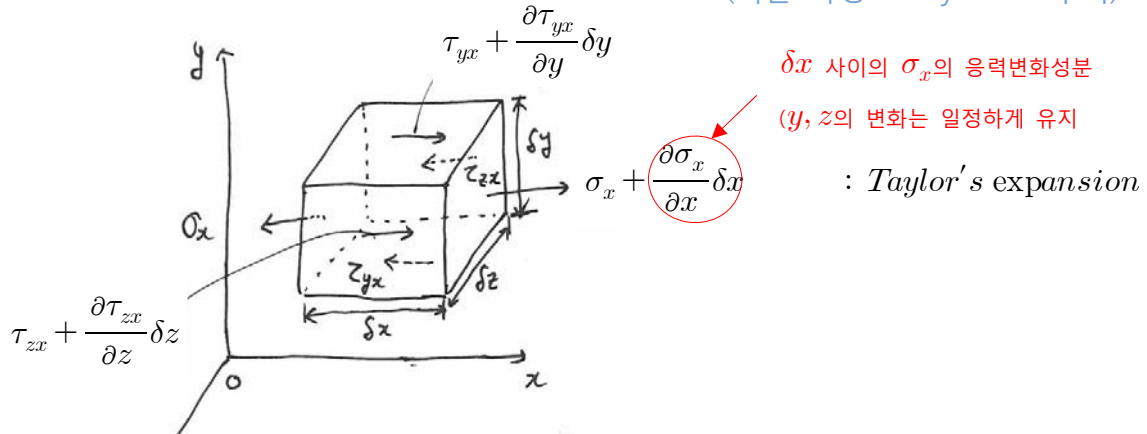
cf). 3D State of stress



2.2 평형방정식

외력을 받고 있는 상태의 물체 내부의 미소체적에 걸리는 힘의 평형

(기본 가정: body force 무시)



→ 6면체의 5개 면 위에서 ⇒ $3 \times 6 \rightarrow$ 18개의 힘이 작용

- x축 방향의 힘의 평형,

$$(\sigma_x + \frac{\partial \sigma_x}{\partial x} \delta x) \delta y \delta z + (\tau_{zx} + \frac{\partial \tau_{zx}}{\partial z} \delta z) \delta x \delta y + (\tau_{yx} + \frac{\partial \tau_{yx}}{\partial y} \delta y) \delta x \delta z - \sigma_x \delta y \delta z - \tau_{zx} \delta x \delta y - \tau_{yx} \delta z \delta x = 0 \quad \text{--- ①}$$

$$\frac{\partial \sigma_x}{\partial x} \delta x \delta y \delta z + \frac{\partial \tau_{zx}}{\partial z} \delta x \delta y \delta z + \frac{\partial \tau_{yx}}{\partial y} \delta x \delta y \delta z = 0 \quad \text{--- ②}$$

x방향의 힘의 평형이 미소체적에서 만족함

$$\therefore \frac{\partial \sigma_x}{\partial x} + \frac{\partial \tau_{yx}}{\partial y} + \frac{\partial \tau_{zx}}{\partial z} = 0 \quad \text{--- ③}$$

- y와 z 방향에서도 힘의 평형 적용,

$$\frac{\partial \tau_{xy}}{\partial x} + \frac{\partial \sigma_y}{\partial y} + \frac{\partial \tau_{zy}}{\partial z} = 0 \quad \text{--- ④}$$

$$\frac{\partial \tau_{xz}}{\partial x} + \frac{\partial \tau_{yz}}{\partial y} + \frac{\partial \sigma_z}{\partial z} = 0 \quad \text{--- ⑤}$$

$$\Rightarrow \frac{\partial \sigma_{ij}}{\partial x_i} = 0 \quad \text{--- ⑥}$$

($x, y, z \rightarrow x_1, x_2, x_3$)

$\begin{pmatrix} \sigma_{xx} \rightarrow \sigma_{11} \\ \sigma_{xy} \rightarrow \sigma_{12} \end{pmatrix}$

- ⑥ 식에서 단위체적 당 질량 f_i (체적력)을 포함하면,

$$\frac{\partial \sigma_{ij}}{\partial x_i} + \rho f_i = 0 \quad (\rho: \text{밀도}) \quad \text{--- ⑦}$$

2.3 모멘트의 평형

* see the section of Equality of shear stress

$$\sigma_{ij} = \sigma_{ji}$$